

Note:

1. The inertia force term changes direction during the piston descend.

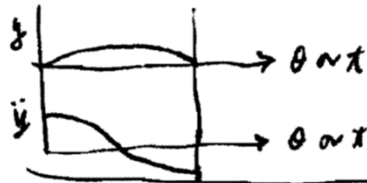
2. The side thrust  $F_s$  is the normal reaction to balance out the x-component of the connecting rod force.

Newton's law  $mi\dot{j} = mg - F_c \cos \phi + \frac{p\pi B^2}{4} - F_f$

Side thrust  $F_s = F_c \sin \phi$

Friction  $F_f = f F_s$   $f$  is friction coefficient

$\theta$  and  $\phi$  related by  $\phi = \sin^{-1}(a \sin \theta / l)$



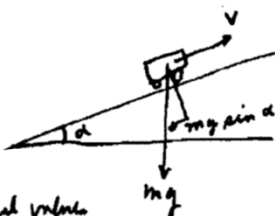
For a 85 mm bore piston at  $p=100$  bar, the pressure force is

$$F_p = \frac{\pi B^2 P}{4} = \frac{\pi (0.085)^2 \times 10^7}{4}$$

$$= 5.67 \times 10^4 \text{ N}$$

$$= 12740 \text{ lb}$$

2.5



typical values

$C_D = 0.4$

$C_x = 0.013$

$m = 1500 \text{ kg}$

$A_v$  (frontal area) =  $2 \text{ m}^2$

$V = 50 \text{ mph} = 22.4 \text{ m/s}$

$\alpha = 15^\circ$

Power required =  $P_{\text{gravity}} + P_{\text{drag}} + P_{\text{friction}}$

$$= [mg \sin \alpha + \frac{1}{2} \rho_a C_D A_v V^2 + C_x mg \cos \alpha] V$$

$$= [1500 \times 9.81 \times \sin 15^\circ + \frac{1}{2} \times 1.2 \times 0.4 \times 22 \times 22 + 0.013 \times 1500 \times 9.81 \times \cos 15^\circ] \times 22.4$$

$$= [3808.5 + 240.8 + 184.8] \times 22.4$$

$$= \underline{95 \text{ kW}}$$

Note: The drag and friction components are roughly equal. The power to propel against gravity is much higher.

Av. acceleration for 40 to 60 mph in 5 sec. (1 mph = 0.447 m/s)

$$a = \frac{\Delta V}{t} = \frac{20 \times 0.447}{5} = 1.79 \text{ m/s}^2$$

$$ma = 1500 \times 1.79 = \underline{2685 \text{ N}} \quad (\underline{603 \text{ lb force}})$$

2.8

$$b_{sfc} = \frac{m_f}{m_f Q_{HV} \eta_f} = \frac{1}{Q_{HV} \eta_f} ; \text{ Put in conversion factors } b_{sfc} \left[ \frac{kg}{kwh} \right] = \frac{1}{Q_{HV} (J/kg) \eta_f} \times 3.6 \times 10^6$$

Fuel	Decoctane	Gasoline	Methanol	H <sub>2</sub>
Q <sub>HV</sub> (low)(J/kg)	44.3 × 10 <sup>6</sup>	44.0 × 10 <sup>6</sup>	20.0 × 10 <sup>6</sup>	120 × 10 <sup>6</sup>
b <sub>sfc</sub> (kg/kwh)	0.271	0.273	0.600	0.100

2.15

1.6 litre displacement engine, 4 cylinder, WOT @ 2500rpm.

Say B/L = 1 ⇒  $\frac{\pi}{4} L^3 = 4000 \text{ cc} \Rightarrow L = 8 \text{ cm}$

mean piston speed  $\bar{S}_p = 2vL = 2 \times \frac{2500}{60} \times 0.08 = \underline{6.67 \text{ m/s}}$

Max piston speed (see fig. 2.2 of Text) = 1.6  $\bar{S}_p = \underline{10.7 \text{ m/s}}$

Max charge velocity at intake = Max piston speed × Area ratio = 10.7 × 5 = 54 m/s

(The following are estimates of the time taken for the various

processes. Refer to figure 1-8 in Text for more precise numbers.)

- Time per cycle @ 2 revolution per cycle =  $\left( \frac{60}{2500} \times 2 \right) = \underline{48 \text{ ms}}$
- Intake, compression, expansion and exhaust each takes up ~ 1/4 of the cycle time ~ 12 ms
- Combustion: Starts ~ 20° BTC, ends ~ 40° ATC. Duration = 60°  
time =  $\frac{60}{720} \times 48 \text{ ms} = \underline{4 \text{ ms}}$
- Flame velocity: Say flame starts at the center (spark plug centrally located). It has to traverse the radius in 4ms  
Flame velocity =  $\frac{0.4 \text{ m}}{4 \times 10^{-3} \text{ s}} = \underline{10 \text{ m/s}}$
- WOT intake "run length" =  $\frac{V_D}{A_{port}} = \frac{V_D}{A_{piston} \times 1/5} = \frac{\text{stroke}}{1/5} = \underline{40 \text{ cm}}$
- Exhaust "run length" =  $\frac{V_D}{A_{port}} \times \text{Temperature ratio} = 40 \text{ cm} \times \frac{425+273}{300} = \underline{93.07 \text{ cm}}$

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2.61 Internal Combustion Engines  
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